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Load and Stress Calculations of Dish Spring

(Reference data: JIS B 2706)

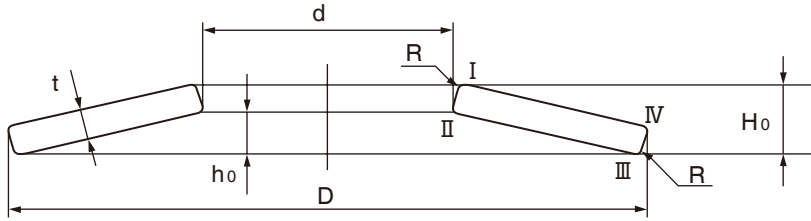


Fig. 3 Dish Spring

D: Diameter of outer periphery (mm)

d: Diameter of inner periphery (mm)

t: Plate thickness (mm)

H₀: Free height (mm)

h₀: Total amount of deflection (H₀ - t) (mm)

E: Longitudinal elastic modulus (N/mm²) (Table 1)

ν: Poisson's ratio of material (0.3)

P: Load (N)

δ: Amount of deflection (mm)

k: Load rate (N/mm)

R: Chamfer radius of corner (mm)

σ_I: Stress on position I (N/mm²)

σ_{II}: Stress on position II (N/mm²)

σ_{III}: Stress on position III (N/mm²)

σ_{IV}: Stress on position IV (N/mm²)

The coefficients used for calculation are as follows:

$$a = \frac{D}{d}$$

$$C_1 = \frac{1}{\pi} \cdot \frac{\left(\frac{a-1}{a}\right)^2}{\frac{a+1}{a-1} - \frac{2}{\ln a}}$$

$$C_2 = \frac{1}{\pi} \cdot \frac{6}{\ln a} \cdot \left(\frac{a-1}{\ln a} - 1\right)$$

$$C_3 = \frac{3}{\pi} \cdot \frac{a-1}{\ln a}$$

Including the correction item $\left(\frac{D-d}{(D-d)-3R}\right)$ that allows for round chamfering of the corner presents the load P by the following formula:

$$P = \frac{D-d}{(D-d)-3R} \cdot \frac{4E}{1-\nu^2} \cdot \frac{t^3}{C_1 D^2} \cdot \delta \cdot \left[\left(\frac{h_0}{t} - \frac{\delta}{t}\right) \cdot \left(\frac{h_0}{t} - \frac{\delta}{2t}\right) + 1 \right]$$

The stresses on the positions I, II, III and IV can be calculated according to the formulas given below. A positive value indicates tensile stress while a negative value indicates compression stress.

$$\sigma_I = \frac{4E}{1-\nu^2} \cdot \frac{t}{C_1 D^2} \cdot \delta \cdot \left[-C_2 \cdot \left(\frac{h_0}{t} - \frac{\delta}{2t}\right) - C_3 \right]$$

$$\sigma_{II} = \frac{4E}{1-\nu^2} \cdot \frac{t}{C_1 D^2} \cdot \delta \cdot \left[-C_2 \cdot \left(\frac{h_0}{t} - \frac{\delta}{2t}\right) - C_3 \right]$$

$$\sigma_{III} = \frac{4E}{1-\nu^2} \cdot \frac{t}{a C_1 D^2} \cdot \delta \cdot \left[(2C_3 - C_2) \cdot \left(\frac{h_0}{t} - \frac{\delta}{2t}\right) + C_3 \right]$$

$$\sigma_{IV} = \frac{4E}{1-\nu^2} \cdot \frac{t}{a C_1 D^2} \cdot \delta \cdot \left[(2C_3 - C_2) \cdot \left(\frac{h_0}{t} - \frac{\delta}{2t}\right) - C_3 \right]$$

The load rate of the spring is non-linear and can be calculated according to the following equation.

$$k = \frac{dP}{d\delta} = \frac{D-d}{(D-d)-3R} \cdot \frac{4E}{1-\nu^2} \cdot \frac{t^3}{C_1 D^2} \cdot \left[\left(\frac{h_0}{t}\right)^2 - 3 \frac{h_0}{t} \cdot \frac{\delta}{t} + \frac{3}{2} \left(\frac{\delta}{t}\right)^2 + 1 \right]$$